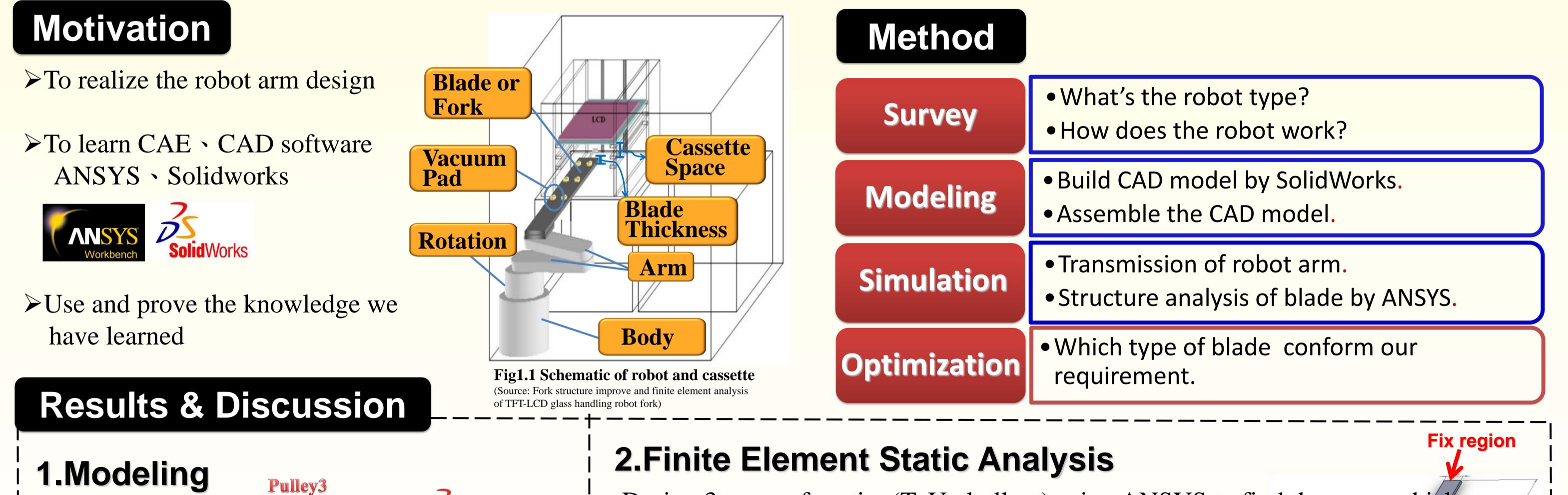
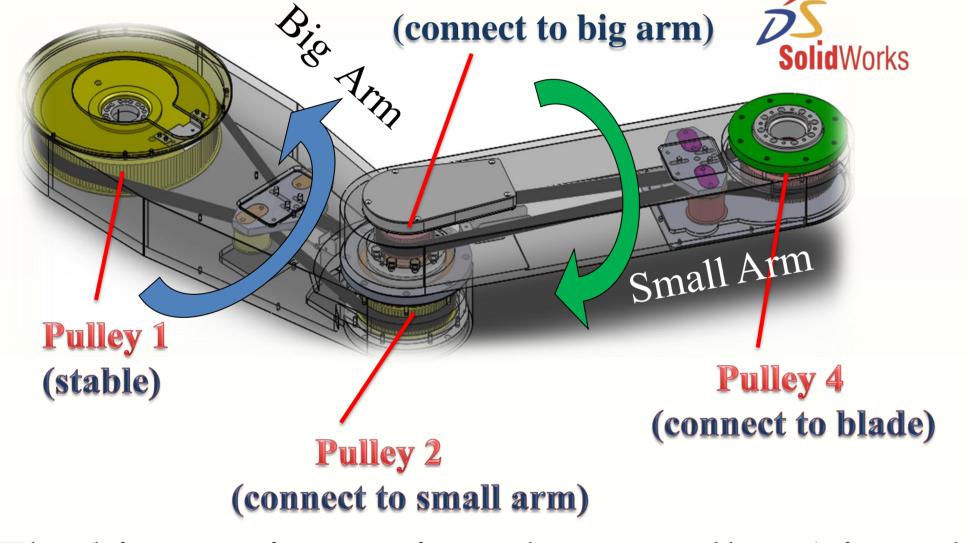
101年大學部國際交流甄選專題成果展 A Study on Robot For TFT LCD **Glass Handling** 5 **Researchers: Po-Chein Chou, Shin-Chuen Peng Advisor:Yi-Cheng Chen**

Introduction

TFT-LCD stands for (Thin Film Transistor Liquid Crystal Display), which is very light, thin, energy saving and low radiation, becoming the most popular display technology now. With the evolution of digital technology, there are more and more demands for large-size glass panels in TFT-LCD manufacturing process. The panel after G6 are already taller than a human beings, so we have to use robot to transport such a large glass panel. In order not to collide the panels during the transportation process, we have to estimate the deformation of loading. After we analyzed the arms deformation, it is strong enough which almost no deformed .Therefore, we focus on the deformation of blade and panel.





The big arm is rotating, due to pulley 1 is stable, pulley 2 and small arm would be anti-rotation. Similarly, pulley 3 is connected to big arm, when the small arm rotate, pulley 4 and blade would be anti-rotation.

4.Optimization of hollow type

The thickness of original design is **4mm**.Try to find the lightest blade which conform our condition.

Design 3 types of section(T, U, hollow), using ANSYS to find the types which conform the safety factor>2 and the smallest deformation under gravity.

Define : Safety Factor= (S-T)/δ

S : Cassette space T : Blade thickness **δ** : Deformation of blade &panel

≻Glass panel size

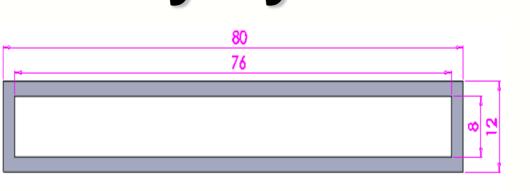
750mm x 620mm x 0.7mm

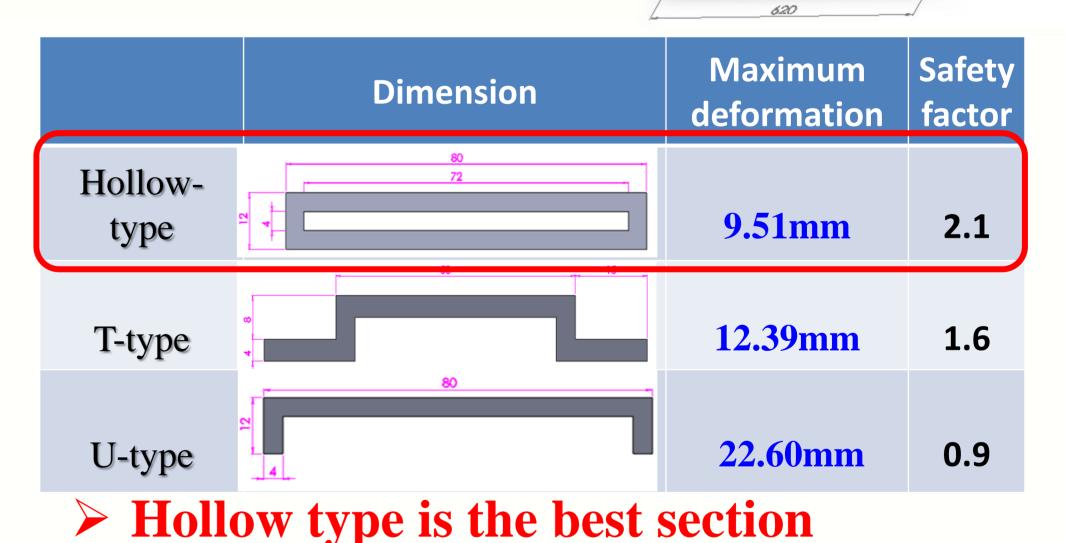
Corning E2K 2370 Kg/m³ Density 69.2 GPa Young's Modulus Poisson's Ratio 0.23 **≻Blade size** :

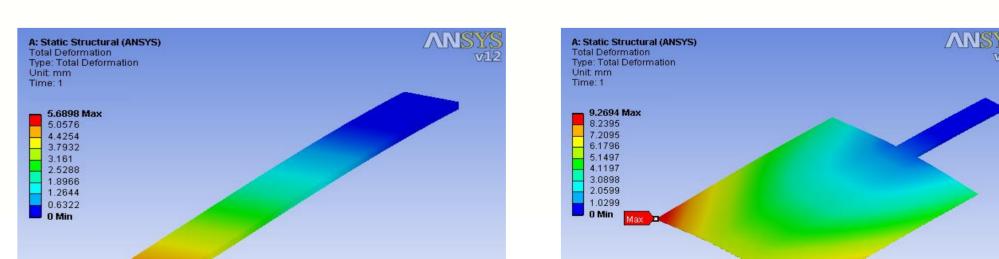
1080mm x 80mm x 12mm Aluminum 7075-T6 2810 Kg/m³ Density

72.0 GPa Young's Modulus 0.33 Poisson's Ratio

5.Verify by ANSYS







We can simplify the loading to 2D case. \rightarrow Cantilever beam Blade

Glass panel→**Distributed loading**

 $\delta_L = \frac{q}{24FI} \left[2L^4 + aL^3 - 6a^2L^2 + (L-a)^4 \right]$

q: distributed load L: bean length a : unloading length E : elastic modulus

Writing a program in Matlab to find the matching value

Fig. 5.1 Dimension of optimization

	Original	Optimization
Thickness	4mm	2mm
Deformation(mm)	9.51	9.27
Weight(kg)	2.17	1.13(52% original)
Table 5.4. Commentions of an invited and antimization		

Table5.1 Comparison of original and optimization

Conclusions

Build the CAD model by SolidWorks

•Realize how does the arm work.

•Analyze different section of blade.

• The best type of blade is hollow type.

•When cassette space is 32mm, the best thickness of blade is 27mm

We have finished the followings:

Fig 5.2 The model without panel

>Maximum deformation **5.68mm** Safety factor : 3.52 >2

Grav

Fig. 5.3 The model with panel >Maximum deformation 9.27mm Safety factor : 2.16 >2